

Oscillatory motion occurs throughout the physical world.

It is universal because systems in stable equilibrium naturally tend to return toward equilibrium no matter how they are displaced.

不但是 universal, 數學描述方式更是相同。

1. 振盪及簡諧運動(SHM)

○ 三個重要量用於描述振盪(oscillation):

(i) 振幅(amplitude): 距離平衡位置的 max. 位移.

(ii) 週期(period, T) = 頻率⁻¹: 週而復始一次的時間, $\therefore [T] = s$

頻率(frequency, f): 1 sec 內, 週而復始的次數,

$\therefore [f] = s^{-1} \equiv Hz$.

(iii) 相位常數(phase constant, 以 ϕ 表示): 起始條件.

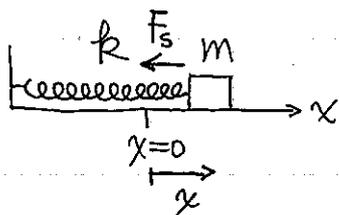
振幅及頻率尚無法完全詳細地描述振盪運動, 如 Fig. 13.2, 有兩同 A 及 f 的 oscillation 可以有不同的函數形式, 此函數形式反應的是回復力(restoring force)的差異.

大部份的物理系統都有相同的回復力形式, 即 ideal spring 的彈力形式。

○ Simple harmonic motion (簡諧運動, 以 SHM 表示)

以 ideal spring 作用力為 restoring force 的振盪運動: SHM.

為大部分真實系統的近似運動, 對小振幅振盪尤其適用。



: mass-spring system with no friction

作用在 m 的力: $F_s = -kx = \text{restoring force}$
回復力

(回復到平衡位置)



m 受外力作用偏離平衡位置 $x=0$, 外力停止作用後, 受到 Spring 的回復力 F_s 作用在 m, m 即進行 SHM。

→ m 的位置函數 $x(t) = ?$ ($x \rightarrow v \rightarrow a, \rightarrow k \rightarrow U \rightarrow E$)

From Newton II: $F_s = ma = m \frac{d^2}{dt^2} x(t) = -kx(t)$

$\therefore m \frac{d^2 x}{dt^2} + kx = 0$

一元二次微分方程的通解 $x(t) = A \cos(\omega t + \phi)$

A 為振幅, $\omega = ?$

$\Rightarrow \frac{d^2 x}{dt^2} = -\omega^2 A \cos(\omega t + \phi) = -\omega^2 x = -\frac{k}{m} x$

$\therefore \omega = \sqrt{\frac{k}{m}} = \text{angular frequency}$

又 from $x(t) = A \cos(\omega t + \phi)$, \cos 的週期為 2π , i.e.

在 $t = T = \text{週期時}$ $\omega T = 2\pi \Rightarrow \omega = \frac{2\pi}{T}$

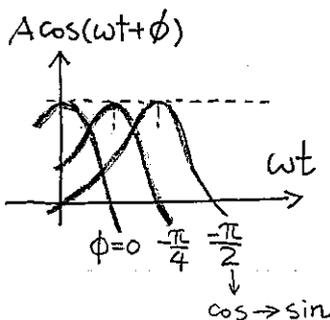
$\therefore [\omega] = \text{rad} \cdot \text{s}^{-1} = 2\pi f$

Note: A 與 f 無關 in SHM, \because restoring force $\propto x$.

If $F_s = -kx$ 不能描述振盪的回復力, 則 f does depend on A, especially, when the displacement x gets too big.

\therefore SHM 僅適用在 J 振幅的振盪運動。

ϕ : phase constant



$\phi = 0$: take max. x at $t = 0$
(在 $t=0$, 將 m 拉到 A)

$\phi > 0$: shift cosine to 左

$\phi < 0$: shift cosine to 右



o SHM 的 v and a

Take $\phi = 0$, i.e. $t=0$, $x(\star) = A$

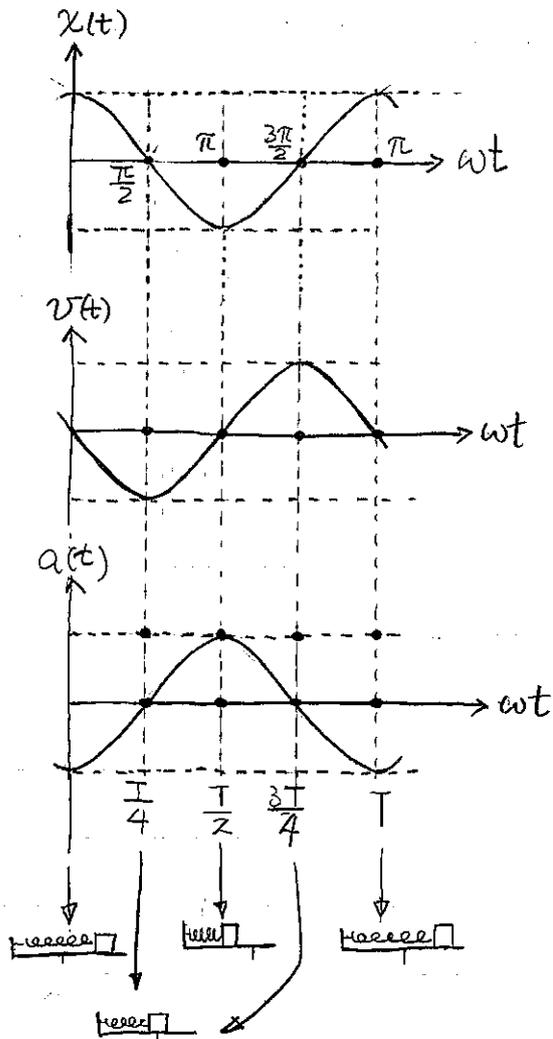
$\therefore x(t) = A \cos \omega t$

$v(t) = \frac{dx(t)}{dt} = -\omega A \sin \omega t = \omega A \cos(\omega t + \frac{\pi}{2})$

$a(t) = \frac{d^2x(t)}{dt^2} = -\omega^2 A \cos \omega t = \omega^2 A \cos(\omega t + \pi)$

$\Rightarrow v(t)$ leads (領先) $x(t)$ 達 $\frac{\pi}{2}$ (i.e. $\frac{T}{4}$)

而 $a(t)$ leads $x(t)$ π , 即 $\frac{T}{2}$ 或可說 lags (落後) $x(t)$ $\frac{T}{2}$.



turning point occurs at $\omega t = n\pi$, i.e. $t = \frac{n}{2}T$

平衡點 (自然長度) 的經過

時間點在 $\omega t = (n + \frac{1}{2})\pi$

$n \in \mathbb{Z}$, i.e. $t = \frac{T}{4}, \frac{3}{4}T, \frac{5}{4}T, \dots$

在 turning point: $v=0$, a 達 max.

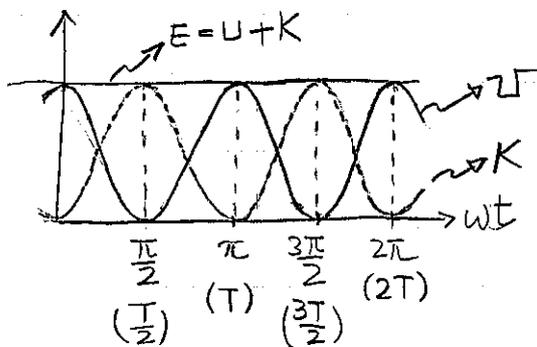
在平衡點: v 達 max, $a=0$.



2. Energy in SHM

$$E = K + U \quad \left\{ \begin{array}{l} K = \frac{1}{2}mv^2 = \frac{1}{2}m\left(\frac{dx}{dt}\right)^2 = \frac{1}{2}m\omega^2 A^2 \sin^2 \omega t \quad (\text{let } \phi=0) \\ U = \frac{1}{2}kx^2 = \frac{1}{2}(m\omega^2)x^2 = \frac{1}{2}m\omega^2 A^2 \cos^2 \omega t \end{array} \right.$$

$$\therefore E = \frac{1}{2}m\omega^2 A^2 = \text{constant} = \frac{1}{2}kA^2 \quad (\text{by } \omega = \sqrt{\frac{k}{m}})$$



⇒ 從能量守恆導出描述運動的微分 eq. (problem 62)

$$E = \frac{1}{2}mv^2 + \frac{1}{2}kx^2 = \text{constant}, \text{ where } x = x(t) \text{ and } v = v(t).$$

$$(i) \frac{dE}{dt} = 0 = m v \cdot \frac{dv}{dt} + kx \cdot \frac{dx}{dt} \\ = m v \frac{d^2x}{dt^2} + kx \cdot v$$

$$\therefore m \frac{d^2x}{dt^2} + kx = 0$$

$$(ii) \frac{dE}{dx} = 0 = \frac{dE}{dt} \frac{dt}{dx} = \frac{dE}{dt} \cdot \left(\frac{dx}{dt}\right)^{-1} = v^{-1} \cdot \frac{dE}{dt} \\ = m \frac{d^2x}{dt^2} + kx$$

∴ Newton II } ⇒ 運動微分 eq.
能量守恆



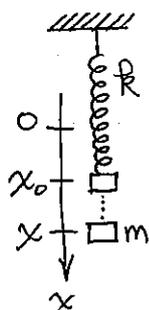
3. SHM 的應用

- (i) 垂直的 spring-mass system
- (ii) Simple pendulum (單擺)
- (iii) torsional oscillator (扭力擺)
- (iv) physical pendulum

(i) Energy 方法

$$E = K + U = \text{constant}$$

$$= \frac{1}{2} m v^2 + \frac{1}{2} k x^2 - m g x$$



$x=0$ = 自然長度
 $x=x_0$ = 有 m 時的平衡位置。
 $\therefore m g = k x_0$

$$\therefore \frac{dE}{dt} = 0 = m v \frac{dv}{dt} + k x \frac{dx}{dt} - m g \frac{dx}{dt}$$

$$\Rightarrow m \frac{dv}{dt} + k x - m g = 0 \quad \therefore \frac{dx}{dt} = v$$

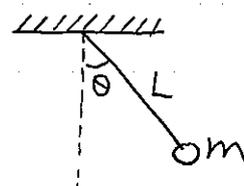
$$\therefore m \frac{d^2 x}{dt^2} + k(x - x_0) = 0, \text{ let } x' = x - x_0, \text{ 則 } \frac{d^2 x}{dt^2} = \frac{d^2 x'}{dt^2}$$

$$\therefore m \frac{d^2 x'}{dt^2} + k x' = 0 \text{ --- SHM.}$$

(ii) 單擺

① Newton II (DIY)

② 能量方法



$$E = K + U = \text{constant}$$

$$= \frac{1}{2} m v^2 + m g L (1 - \cos \theta) \quad (U=0 \text{ at the lowest point})$$

$$= \frac{1}{2} m (L \Omega)^2 + m g L (1 - \cos \theta), \text{ where } \Omega = \text{angular speed} = \frac{d\theta}{dt}$$

$$= \frac{1}{2} m L^2 \left(\frac{d\theta}{dt} \right)^2 + m g L (1 - \cos \theta)$$

$$\frac{dE}{dt} = 0 = m L^2 \left(\frac{d\theta}{dt} \right) \cdot \frac{d^2 \theta}{dt^2} + m g L \sin \theta \cdot \frac{d\theta}{dt}$$

$$\therefore L \frac{d^2 \theta}{dt^2} + g \sin \theta = 0$$

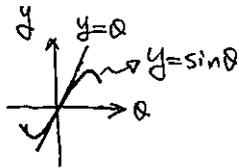
for $\small \Delta$, 角度擺動, i.e. $\theta \ll 1$, then $\sin \theta \approx \theta$

$$\therefore L \frac{d^2 \theta}{dt^2} + g \theta = 0 \text{ (運動微分方程)} \quad (\sim m \frac{d^2 x}{dt^2} + k x = 0)$$

$$\therefore \omega = \sqrt{\frac{g}{L}} \quad \text{and } T = 2\pi \sqrt{\frac{L}{g}} \text{ (} \small \Delta \text{ } m \text{ 及 } \small \Delta \text{ 擺幅無關)} \quad \text{羅}$$

小角度到底要多小? i.e. θ 要多小, $\sin\theta \approx \theta$?

Check: θ (度)	57.3°	28.65°	11.46°	5.73°	2.87°
θ (rad.)	1	0.5	0.2	0.1	0.05
$\sin\theta$	0.84	0.48	0.199	0.0998	0.05
$\frac{ \theta - \sin\theta }{\theta}$	16%	4%	0.5%	0.2%	0

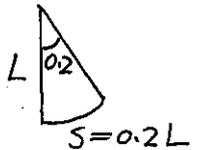


$$\sin\theta = \theta - \frac{\theta^3}{3!} + \frac{\theta^5}{5!} - \frac{\theta^7}{7!} + \dots$$

$\Rightarrow \theta \leq 0.2 \text{ rad}$, $\sin\theta \approx \theta$, i.e. $s \leq 0.2L = 20\% L$

\therefore 相同擺幅之下 (i.e. s 相同)

L 愈大, 擺鐘愈準。



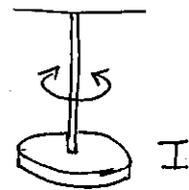
(iii) Torsional oscillation

為轉動的 Hooke's law

$$E = K + U$$

$$= \frac{1}{2} I \omega^2 + \frac{1}{2} k \theta^2, \text{ where } k = \text{torsional constant}$$

and $\frac{d\theta}{dt} = \omega$ $\theta = \text{angular displacement}$



$\therefore E$ 的公式 $\sim \frac{1}{2} m v^2 + \frac{1}{2} k x^2$ in ideal spring system

$$\therefore \omega = \sqrt{\frac{k}{I}}$$

Note: $U(\theta) \Rightarrow -\frac{dU(\theta)}{d\theta} = \tau(\theta) = -k\theta, \therefore U(\theta) = \frac{1}{2} k \theta^2 + \text{constant}$.



(iv) physical pendulum

支點不在CM的不規則物體

$$\tau = I\alpha \text{ and}$$

$$\vec{\tau} = \vec{L} \times (m\vec{g})$$

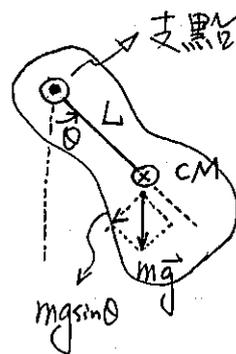
$\vec{\tau}$ 的方向 \otimes 且 $\vec{\theta}$ 的方向相反

$$\therefore \tau = -mgL \sin\theta \approx -mgL\theta \text{ (for small } \theta)$$

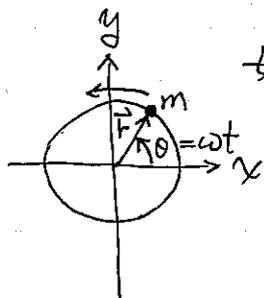
$$= I\alpha = I \frac{d^2\theta}{dt^2}$$

$$\therefore I \frac{d^2\theta}{dt^2} + mgL\theta = 0 \Rightarrow \omega = \sqrt{\frac{mgL}{I}}, T = 2\pi \sqrt{\frac{I}{mgL}}$$

測量 T 可得 I , 再由平行軸定理可得 I_{CM} .



4. 等速率圓周運動 (UCM) vs. SHM



如左圖的 UCM, 以 m 的 \vec{r} 與 x -axis 的夾角為 θ ,

$$\text{若 } m \text{ 以 } \omega \text{ 作 ccw 的 UCM 則 } \theta = \omega t$$

$$\Rightarrow \left. \begin{array}{l} \vec{r} \text{ 的 } x \text{ component 為 } x(t) = r \cos \omega t \\ y \text{ component 為 } y(t) = r \sin \omega t \end{array} \right\} \text{ 相差 } 90^\circ$$

$\therefore \vec{r}$ 的 x, y 分量皆為 SHM (SHM 是 UCM 在軸上的投影量)

\Rightarrow understand ω in SHM even though there is no angle involved.



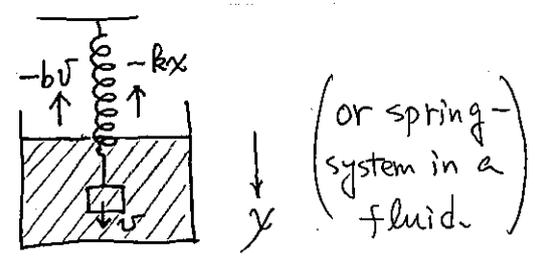
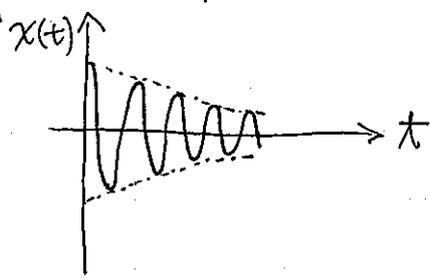
5. Damped and driven 振盪運動

o Damped (阻滯) harmonic motion

In SHM: $\frac{d^2x}{dt^2} + \omega^2 x = 0$ is an ideal case in which no energy dissipation.

But in reality, 一定有能量損耗, 如果沒有額外的 energy 補入系統, 則振幅將慢慢減小, 最後終致停止.

⇒ damped oscillation



一般的 damping force (阻滯力, 用 F_d 表示) $\propto v$ 且 \perp 運動方向相反,

$$\therefore F_d = -bv = -b \frac{dx}{dt}$$

where $b > 0$ and is \sim damping strength

$$\therefore \text{Newton II: } F_{net} = ma = F_s + F_d = -kx - b \frac{dx}{dt}$$

$$\text{i.e. } m \frac{d^2x}{dt^2} + b \frac{dx}{dt} + kx = 0$$

其解為 $x(t) = A e^{-bt/2m} \cos(\omega' t + \phi)$, where

$$\omega' = \sqrt{\frac{k}{m} - \left(\frac{b}{2m}\right)^2} = \sqrt{\omega_0^2 - \left(\frac{b}{2m}\right)^2}$$

$\omega_0 = \sqrt{\frac{k}{m}}$ = natural angular-frequency.

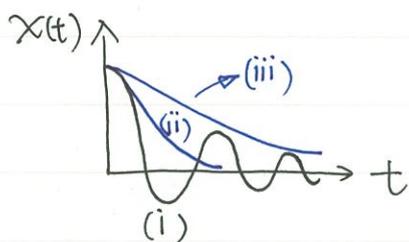
ω' 決定 $x(t)$ 的行為

(i) $\omega' \in \text{實數}$ i.e. $\omega_0 > \frac{b}{2m}$, 阻滯強度小時 \Rightarrow 振幅衰減 + 振盪
如上左圖, 此時為 underdamped.



(i) 阻滯強度增加到 $\omega_0 = \frac{b}{2m}$, 則 $x(t) = A \cos \phi \cdot e^{-bt/2m}$
 \Rightarrow 沒有振盪, 且快速到 $x=0$: critical damping.

(ii) 阻滯強度更大, 阻滯力主宰運動, 沒有振盪且緩慢衰減到 $x=0$



$\omega' \in$ pure imaginary.
 (純虛數)

(i) underdamping
 (ii) critical damping
 (iii) overdamping

o Driven oscillation (強迫振盪) and resonance (共振)

強迫振盪:

\vec{F}_{ext} on oscillation: driven oscillation (外力作功, 輸入能量)

for mass-spring system: $F_{ext} = F_0 \cos \omega_d t$, where

$\omega_d =$ driving frequency

$$\begin{aligned} \text{Newton II: } F_{total} &= F_s + F_d + F_{ext} \\ &= -kx - b \frac{dx}{dt} + F_0 \cos \omega_d t \\ &= ma \\ &= m \frac{d^2x}{dt^2} \end{aligned}$$

$$\therefore m \frac{d^2x}{dt^2} + b \frac{dx}{dt} + kx = F_0 \cos \omega_d t$$

剛開始的運動較複雜, 但最後會達到 steady-state 振盪, \therefore

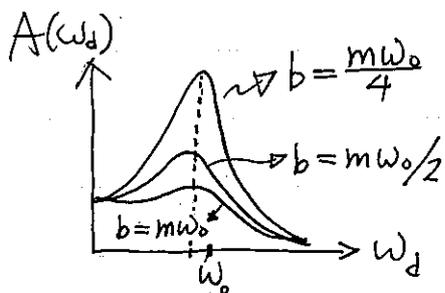
We expect $x(t) = A \cos(\omega_d t + \delta)$ for steady-state. 代入微分運動

吧. $\Rightarrow A(\omega_d) = \frac{F_0}{m} \left[(\omega_d^2 - \omega_0^2)^2 + \frac{b^2}{m^2} \omega_d^2 \right]^{-1/2}$

$$A \text{ has max when } \omega_d^2 = \omega_0^2 - \frac{b^2}{2m^2}$$

i.e. $A(\omega_d)$ has max. when ω_d is near ω_0 : Resonance.





the weaker the damping,
the more sharply peaked is the
resonance curve.

∴ 在弱阻滯系統可用小區動力達到大幅共振。

實際建物應避免產生共振，如建築物的 ω_0 應避開地震波的頻率。

Resonance is important in microscopic systems:

magnetron \rightarrow microwaves heat food
ionize gases.

CO_2 acting like mass-spring system resonates at some of
the frequencies of IR. \rightarrow greenhouse effect.

