

In this homework, you are going to design a PID controller to control the inverted pendulum model in homework 4.

1) Use ode45 to simulate the output response in time-domain.

## Problem 1

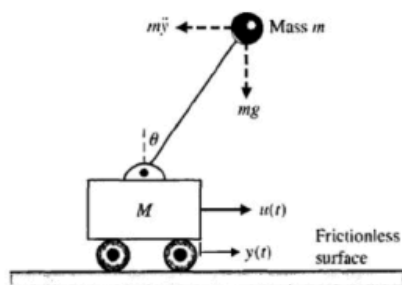


Figure 1 A cart and an inverted pendulum.

Force (horizontal):

$$(M + m)\ddot{y}(t) = u(t) - (mL\ddot{\theta}(t) \cos \theta(t) - mL\dot{\theta}(t)^2 \sin \theta(t))$$

Torque:

$$mL^2\ddot{\theta}(t) = mgL \sin \theta(t) - mL\ddot{y}(t) \cos \theta(t)$$

Assume that  $M=100\text{Kg}$ ,  $m=10\text{Kg}$ ,  $L=1\text{m}$ ,  $g=9.81\text{m/s}^2$  and  $G_c(s) = K_p + \frac{K_I}{s} + sK_D$ ,

\*This is a guideline for Problem 1.

\*The equations are not exactly the same, but same concepts

1) Use ode45 to simulate the output response in time-domain.

$$\ddot{y} = -\frac{mg}{M}\theta + \frac{1}{M}u$$

$$\ddot{\theta} = \frac{(M+m)g}{ML}\theta - \frac{1}{ML}u$$

$$x_1 = y, x_2 = \dot{y}, x_3 = \int \theta dt, x_4 = \theta, x_5 = \dot{\theta}$$

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = -\frac{mg}{M}x_4 + \frac{1}{M}u$$

$$\dot{x}_3 = x_4$$

$$\dot{x}_4 = x_5$$

$$\dot{x}_5 = \frac{(M+m)g}{ML}x_4 - \frac{1}{ML}u$$

$$\text{PID control } U(s) = G_c(s)(R(s) - \theta(s))$$

$$u(t) = -K_p\theta(t) - K_I \int \theta(t)dt - K_D \frac{d\theta(t)}{dt} = -K_p x_4 - K_I x_3 - K_D x_5$$

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = -\frac{mg}{M}x_4 + \frac{1}{M}(-K_p x_4 - K_I x_3 - K_D x_5)$$

$$\dot{x}_3 = x_4$$

$$\dot{x}_4 = x_5$$

$$\dot{x}_5 = \frac{(M+m)g}{ML}x_4 - \frac{1}{ML}(-K_p x_4 - K_I x_3 - K_D x_5)$$

$$x_{10} = 0, x_{20} = 0, x_{30} = 0, x_{40} = 1, x_{50} = 0$$

q(s) can be obtained by (sl-A).

$$q(s) = MLs^3 - K_D s^2 - [(M+m)g + K_p]s - K_I$$

$$G(s) = \frac{-0.01}{(s + 3.285)(s - 3.285)}, \text{PID}(s) = \frac{K_D s^2 + K_P s + K_I}{s} = \frac{K_D (s - z_1)(s - z_2)}{s}$$

$$q(s) = MLs^3 - K_D s^2 - [(M + m)g + K_P]s - K_I = 0$$

#For Stable system,  $-K_D > 0$ ,  $-[(M + m)g + K_P] - \frac{K_I}{K_D} ML > 0$ ,  $-K_I > 0$

#For Marginally Stable system,  $K_D < 0$ ,  $K_I < 0$ ,  $K_P = -[(M + m)g] - \frac{K_I}{K_D} ML$

#I design a stable system with  $K_D = -50$ ,  $K_I = -50$ ,  $K_P = -1229.1$

```
close all; clear;
fprintf("Q1 \n");
```

Q1

```
% Parameters
```

```
m = 10; M = 100; L = 1; g = 9.81;
```

```
% PID tuning
```

```
KD = -50; KI = -50; KP = -((M+m)*g)-((KI/KD)*M*L)-50;
```

```
% Reference signal
```

```
r = 0; c_in = [0;1/M;0;0;-1/(M*L)];
```

```
ref_sig=@(t, x)[x(2);-m*g*x(4)/M;x(4);x(5);(M+m)*g*x(4)/(M*L)]+(-KP*x(4)-KI*x(3)-KD*x(5))*c_in;
```

```
T = linspace(0, 45, 3*1e3);
```

```
x0 = [0; 0; 0; 1; 0];
```

```
[t, Y] = ode45(@(t, x) ref_sig(t, x), T, x0);
```

```
u = (-KP.*Y(:,4)) + (-KI.*Y(:,3)) + (-KD.*Y(:,5));
```

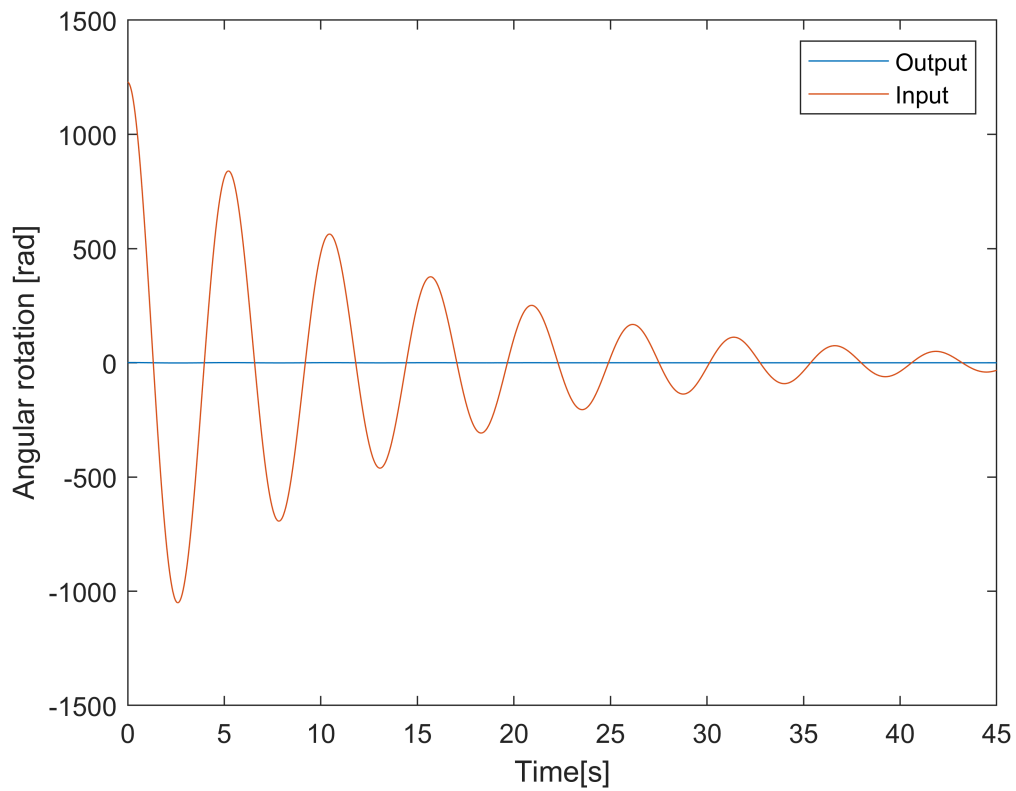
```
% Plot
```

```
figure; plot(t, Y(:,4), t, u);
```

```
xlabel('Time[s]');
```

```
ylabel('Angular rotation [rad]');
```

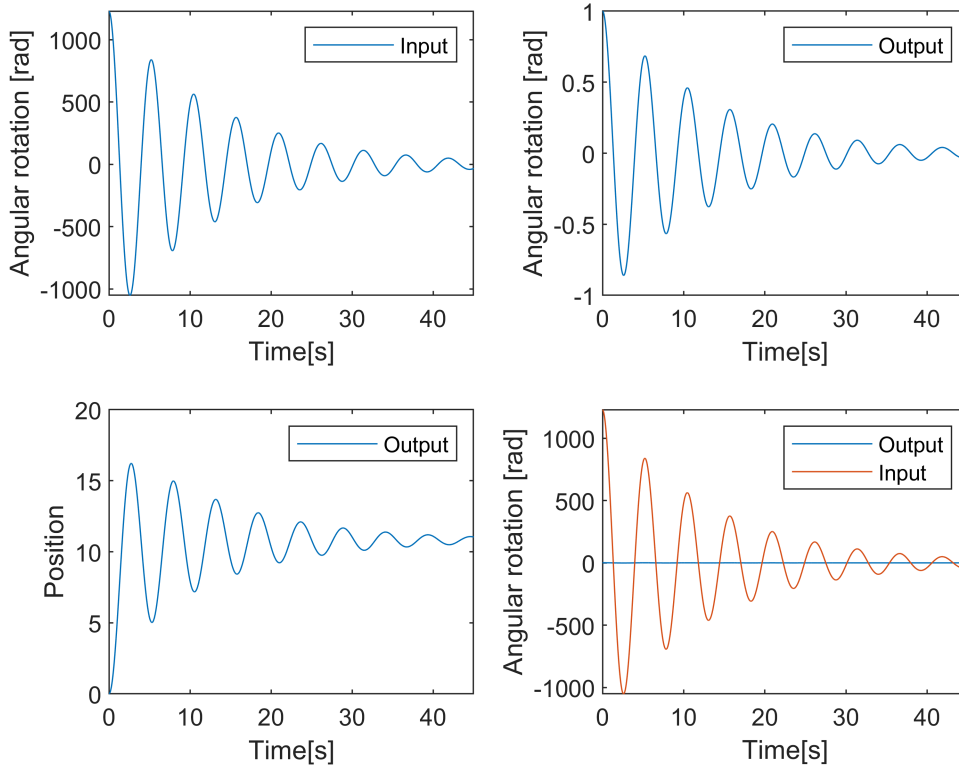
```
legend(["Output"], ["Input"]);
```



```

figure;
subplot(2,2,1); plot(t, u);
xlabel('Time[s]'); ylabel('Angular rotation [rad]');
legend(["Input"]);
subplot(2,2,2); plot(t, Y(:,4));
xlabel('Time[s]'); ylabel('Angular rotation [rad]');
legend(["Output"]);
subplot(2,2,3); plot(t, Y(:,1));
xlabel('Time[s]'); ylabel('Position');
legend(["Output"]);
subplot(2,2,4); plot(t, Y(:,4), t, u);
xlabel('Time[s]'); ylabel('Angular rotation [rad]');
legend(["Output"], ["Input"]);

```



```

syms x;
eqn = M*L*x^3 - KD*x^2 - ((M+m)*g+KP)*x - KI == 0;
s = solve(eqn, x);
digits(6) % sets the precision used by vpa to 6 significant decimal digits.
vpa(s) % Variable-precision arithmetic

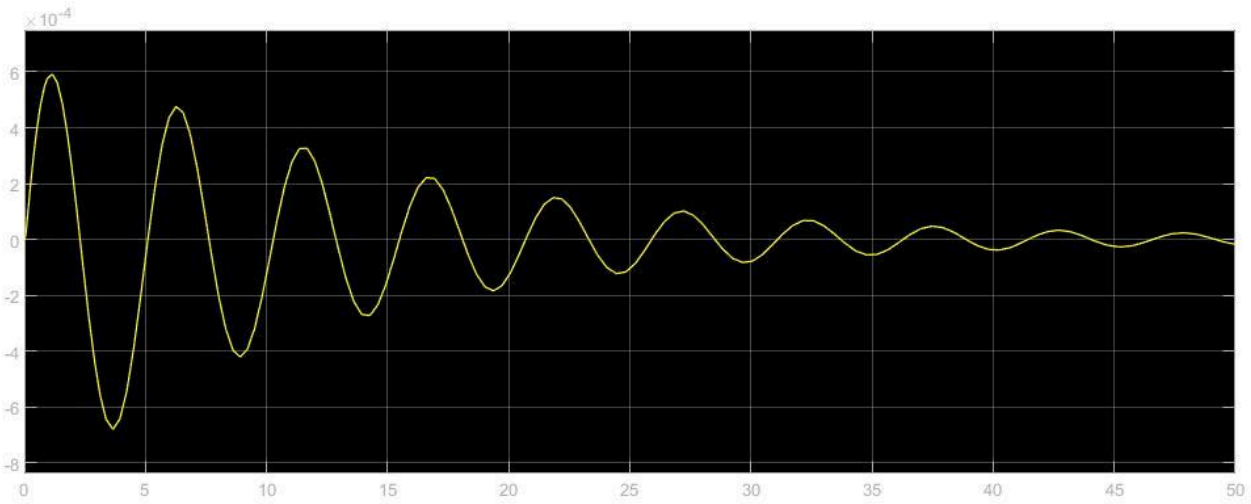
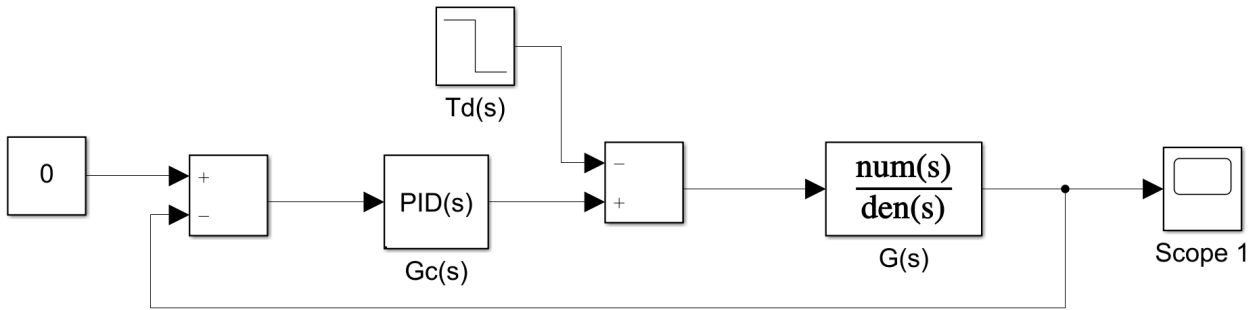
```

ans =

$$\begin{pmatrix} -0.345627 \\ -0.0771863 - 1.20029i \\ -0.0771863 + 1.20029i \end{pmatrix}$$

由模擬結果顯示，output 確實會漸漸 decay 至 0，這顯示此為一個穩定的系統。

2) Use Simulink to simulate the output response in s-domain.



由 Simulink 模擬結果顯示，此為一個穩定的系統。

```
close all; clear;
fprintf("Q2 \n");
```

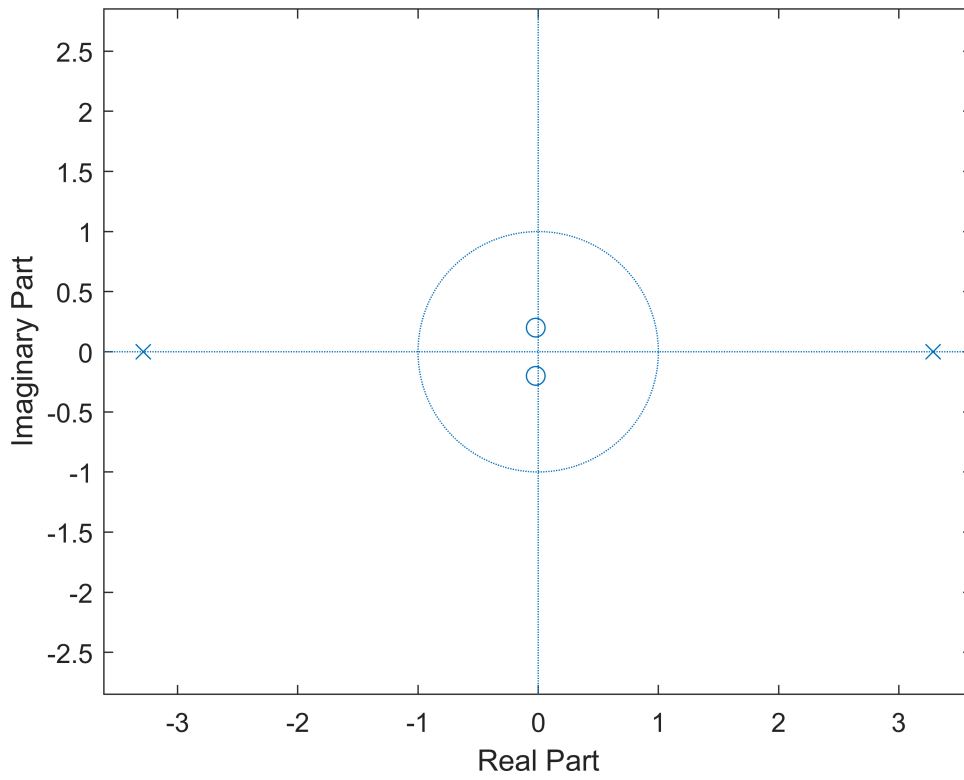
Q2

```
% Parameters
m = 10; M = 100; L = 1; g = 9.81;

% PID tuning
KD = -50; KI = -50; KP = -((M+m)*g)-((KI/KD)*M*L)-50;

b = [-1/(M*L)]; % -0.01
a = [1 0 -(M+m)*g/(M*L)]; % [1 0 -10.7910]

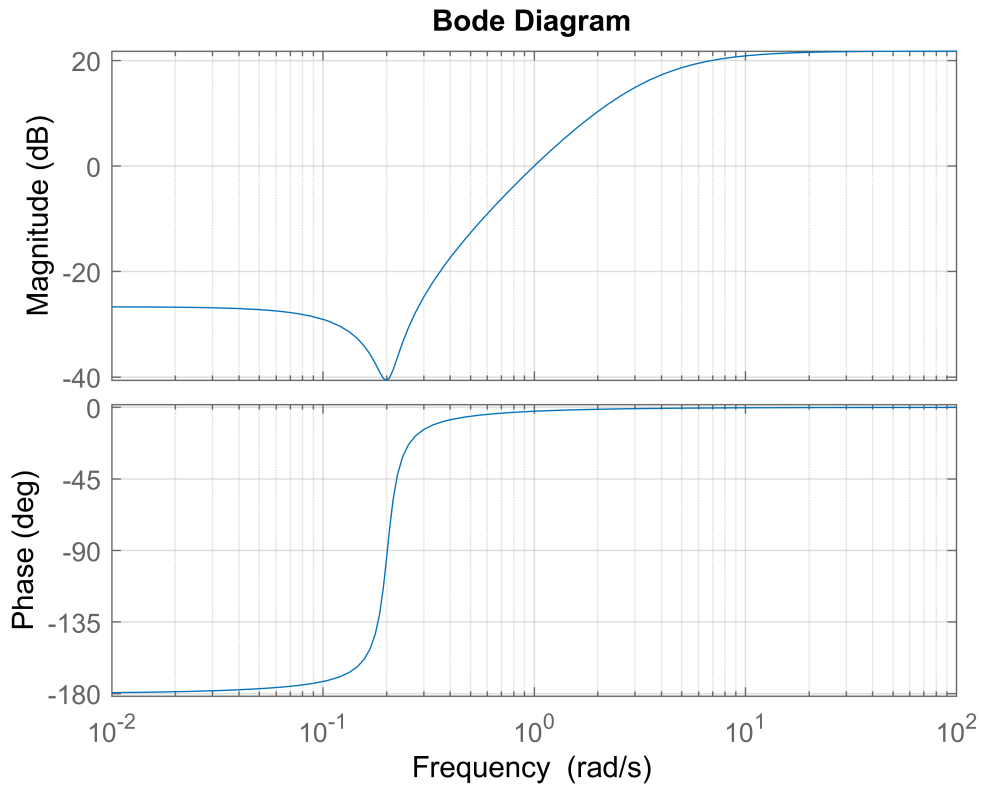
b_pid = [KP KI KD]; a_pid = [1];
b_all = conv(b_pid, b); a_all = conv(a_pid, a);
figure; zplane(b_all,a_all);
```



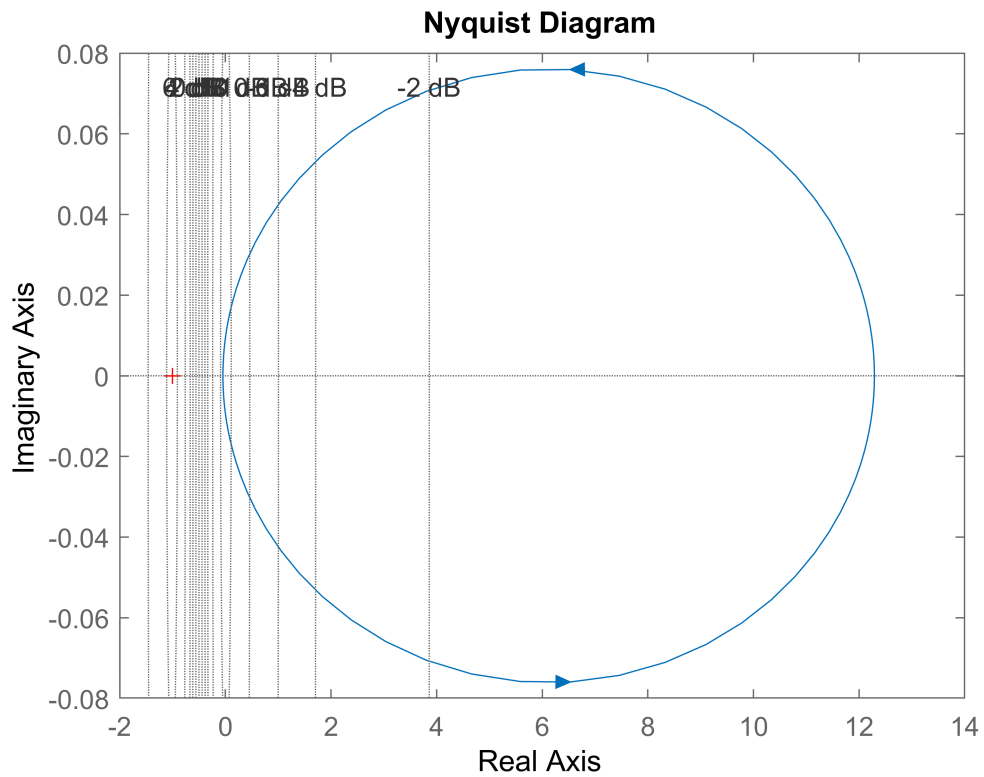
```
% open-loop system  
fprintf("open-loop system \n");
```

open-loop system

```
sys = tf(b_all, a_all);  
figure; bode(sys); grid on;
```



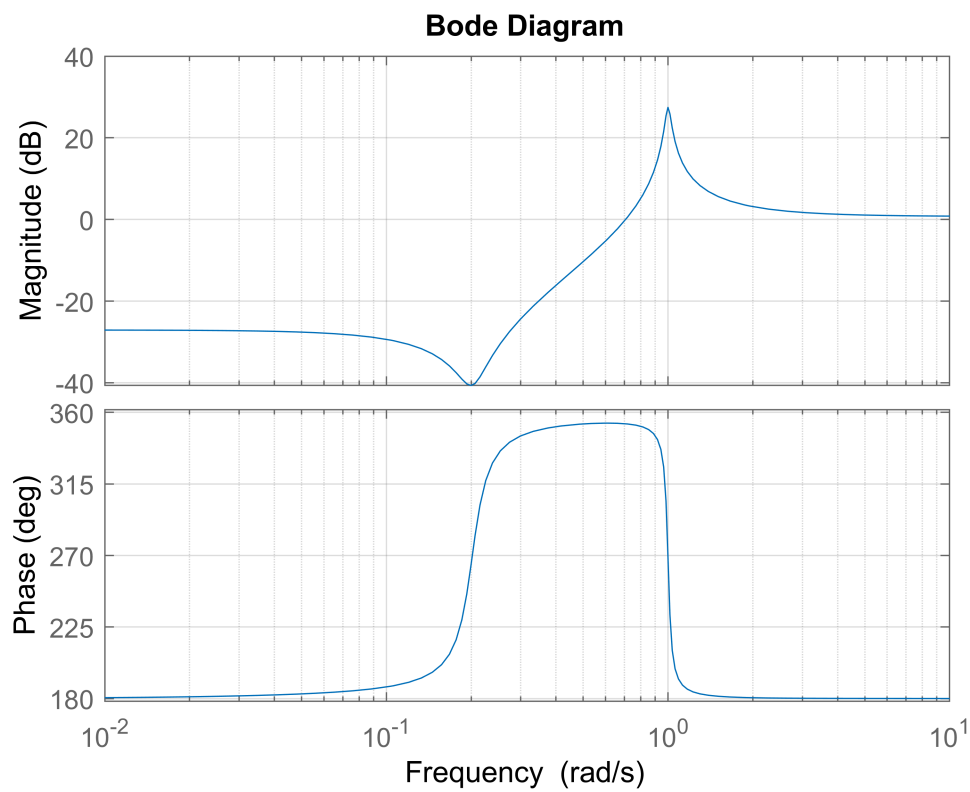
```
figure; nyquist(sys); grid on;
```



```
% closed-loop system
fprintf("closed-loop system \n");
```

closed-loop system

```
T_sys = feedback(sys,1,+1);
figure; bode(T_sys); grid on;
```



```
figure; nyquist(T_sys); grid on;
```



